IN THE SPECIFICATION

Please revise the cited specification paragraphs as follows:

[2] Current unitized seal designs are relatively complicated assemblies which seal rotational interfaces, such as where a yoke exits/enters an axle housing. Conventional seal designs are implemented to reduce the difficulty of assembly and to eliminate the requirement to change yokes after a seal failure. A unitized design includes rubber internal bump stops to preload excluder lips during yoke installation, and a running surface sleeve for a main seal lip interface. The bump stop and running sleeve interaction may create seal assembly and longevity issue tradeoffs.

[9] The fixed portion includes a press fit outer diameter which is pressed into an opening located within the housing assembly. A seal mount segment located transverse to the axis of rotation receives a resilient annular seal which rides upon the running sleeve and the upper seal segment.

[17] Figure 1 illustrates a general side view of an axle housing assembly 10 having a forward drive yoke 12 and a rear~~-~~ward drive yoke 14 which extend there~~-~~from for rotation about an axis A. The assembly 10 drives a vehicle axle 15 such as that utilized by a heavy vehicle as generally known. It should be understood that although a particular component arrangement is disclosed in the illustrated embodiment, other arrangements can be utilized with ~~will benefit from~~ the instant invention.

[18] The forward drive yoke 12 passes through a forward unitary input seal assembly 16 (also illustrated in Figure 2). The rearward drive yoke 14 passes through a rearward unitary output seal assembly 18 (also illustrated in Figures 3). It should be understood that relative positional terms such as "forward," "aft," "upper," "lower," "above," "below," and the like are with reference to the normal operational attitude of the vehicle and should not be considered

otherwise limiting.

[20] The rotating portion 20 includes a running sleeve 24, a slinger segment 26 and an upper seal segment 27. The running sleeve 24 and upper seal segment 27 are located generally parallel to the axis A while the slinger segment 26 is arranged perpendicular thereto. The upper seal segment 27 is a U-shaped portion which extends back from the slinger segment 26. The upper seal segment 27 extends parallel to and is spaced away from the running sleeve segment 24 to form an annular seal cavity 29.

[21] The slinger segment 26 radially extends outward beyond of the fixed portion 22 so as to protect the fixed portion 22. A slinger end 26Aa of the slinger segment 26 is preferably raked in the direction of airflow (illustrated by schematically by arrow F) to further minimize the ingestion of debris in to the seal assembly 16. A sSlinger extension 26Bb on fixed portion 22 is adjacent slinger end 26Aa. Slinger extension 26Bb preferably does not extend radially outward as far much as slinger end 26Aa. Slinger end 26Aa extends beyond slinger extension 26Bb for airflow protection Slinger extension 26Bb is preferably of a diameter larger than housing assembly 10 to prevent runoff of debris falling into the seal assembly 16. With a larger diameter, the runoff will "roll" circumferentially around the sides and not fall into the seal assembly 16.

[22] The running sleeve 24 includes a resilient seal 31 mounted therein to operate as a static seal. Resilient seal 31 preferably provides for static sealing. Preferably, metal-on-metal contact is provided with only a short interval of resilient materials for static sealing. It should be understood that various seal arrangements in this location can be utilized with will benefit from the present invention. The rotating portion 20 is preferably assembled with a press fit onto the yoke 12 to contact a yoke shoulder S at surface 27E, however, other assembly procedures can will also be utilized with benefit from the present invention.

[23] The running sleeve 24 is preferably arranged to contact the yoke 12. Contact between an end of the running sleeve ~~end 24a~~ and the yoke 12 provides a metal to metal contact which operates as a thermal transfer path away from a resilient annular seal 36. The fixed portion 22 includes a press fit outer diameter 32 which is pressed into an opening 35 located within the housing assembly 10. The press fit outer diameter 32 is generally parallel to an outer surface 27A~~a~~ of the upper seal segment 27. The outer diameter 32 is generally parallel to the axis A and terminates with a seal mount segment 34 which is generally perpendicular thereto. ~~That is, T~~the press fit outer diameter ~~segment~~ 32~~;~~ and the seal mount segment 34 form a generally question mark like shape in cross-section.

[24] The seal mount segment 34 receives the resilient annular seal 36. The resilient annular seal 36 includes a groove 38 within which the seal mount segment 34 engages to locate the resilient annular seal 36 adjacent the seal cavity 29.

[25] The annular seal 36 preferably includes a multiple of wipers 40 which ride upon the running sleeve 24. The wipers 40 form a labyrinth to prevent debris from entering through the seal assembly 16. A reverse wiper 42 extends from the resilient annular seal 36 to engage the upper seal segment 27. The resilient annular seal 36 may additionally include a retainer channel 44 to receive an annular retainer 46 such as an annular spring. It should be understood that a multiple of seal geometries ~~will benefit from~~ can be utilized with the present invention.

[26] Referring to Figure 5, the seal assembly 18 is illustrated in cross-section and is of a construction as described above with regard to seal assembly 16. As the seal assembly 18 is rearward facing relative the housing assembly 10 (Figure 1), a slinger end 26A~~a~~' of a slinger segment 26' of seal assembly 18 is raked in the direction of airflow (illustrated by schematically by arrow F and opposite that of seal assembly 16 to further minimize the ingestion of debris in~~to~~

the seal assembly 18. Such an integral slinger arrangement provides for a slinger in heretofore unattainable locations as the slinger is internal to the rotating portion of the seal assembly 18.

[27] Slinger end 26A~~a~~' extends outward of slinger extension 26B~~b~~'. Preferably slinger end 26A~~a~~' is angled radially away from centerline A to operate as a drip edge for debris which thereby travel circumferentially around the seal assembly 18 rather than into the seal assembly 18.

Please add a new paragraph [28] as follows:

[28] The slinger end 26A and the slinger extension 26B both extend generally conically, as is clear from Figures 4 and 5. The slinger end 26A and slinger extension 26B extend in a direction having a component both radially outwardly of the axis, and along the axis. Further, as can be seen, one of the slinger end 26A and slinger extension 26B is spaced in the direction of air flow, and that one extends radially outwardly of the other.